# Turbo gear drive systems





## Turbo-Helical Gear Units

References: more than 1,000 units

	Single-helical gear drive TSG 1
Size	250 up to 1,050
Motor rating P (kW)	100 up to 4,400
Motor speed n <sub>1</sub> (rpm)	1,000 up to 3,600
Output speed n <sub>2</sub> (rpm)	4,500 up to 37,400
Bearings fitted	plain journal bearings
Used for	oil and gas industry, compressor drive



٨

C B

and the second se	Single-helical gear drive TSG 2
ze	160 up to 800 (12 sizes available)
Notor rating P (kW)	370 up to 30,000
Notor speed n <sub>1</sub> (rpm)	3,500 up to 15,000
Output speed n <sub>2</sub> (rpm)	1,000 up to 4,950
earings fitted	plain journal bearings
sed for	power generation plant, steam turbine





#### **General Specification**

The series "Turbo-Helical Gear Units" includes gear units in the range of centre distance of a = 160 mm up to 800 m. The centre distances have been standardized according to standard numbers. The ratio range is between i = 1 and i = 8. Larger centre distances, larger ratios as well as centre distances deviating from the series of standard numbers can be realized as spezial designs on request.

#### Gear Unit

The gear units of this series can be designed as:

- gear units with single-helical gearing and end-thrust bearings on the pinion and gear shaft (tapered land or tilting pad design),
- gear units with double-helical gearing or single-helical gearing with thrust collar and gear side end-thrust bearing (tapered land or tilting pad design),
- special design: Gear units with single-helical gearing, with gear hollow shaft, and integrated torsional shaft as well as end-thrust bearings on the pinion and gear shaft (tapered land or tilting pad design),
- special design: Gear units with single-helical gearing, with pinion and gear hollow shaft, and integrated torsional shafts as well as gear side end-thrust bearing (tapered land or tilting pad design).

The housings of the gear units are designed in three different ranges depending on the ratio.

#### Housing

Torsionally rigid housings with massive ribs consist of cast iron. They are so designed as to minimize vibrations and to ensure a very silent running. The housings can be designed as welded construction on costumers request. From a centre distance of a = 560 upwards, the housings are equipped with an oil sump in order to limit the vertical expansion caused by the oil heating. All housings have got inspection apertures, exausters, lifting facilities , and mounting surfaces for alignment.

#### Gearing

The pinion shafts and gears consist of alloyed case-hardening steels, the gearing is case-hardened and ground as well as optimized with regard to its geometry according to the respective use. In case of need, the gearings are corrected in order to compensate sags/deformations and to optain an optimum contact pattern under full load conditions.

#### Bearing

For the shaft bearing there are usually used heavy-duty sliding bearings divided into two. Here circular cylindrical, lobe- or tilting pad type bearings are intended depending on the operating conditions. The axial forces are taken by axial collars on the radial bearings or by separate end-thrust bearings.

#### Lubrication and Cooling

In principle there has been intended a pressure circulating lubrication in order to supply points of tooth engagement and bearings as well as to ensure the heat removal. The pressure circulating lubrication can be realized by flanged oil pump, separate lube-system or by connecting the gear unit to the available supply systems.

#### Shafts

Non-toothed shafts are made of steel highly hardened and tempered. The basic units are dynamically balanced according to the requirements on the running qualities of the turbo-helical gear units. The shaft journals can be supplied as a cylindrical, taper or a flange design.

#### Monitoring

The gear units are equipped with machine thermometers attached to the bearings according to standard. The furnishing of the gear units with further monitoring devices or the establishment of connection facilities for control of the oil pressure, oil temperature, bearing temperature, shaft vibration, axial vibrations etc. are possible.

#### Design

In particular the rationally symmetrical components are specifically optimized for the respective use by means of the application of CAD. For the gearing design there can be fulfilled the general national and international regulations of design (DIN, AGMA, API, classification societies etc.). The design of the sliding bearings is carried out on the basis of the theory of hydrodynamic lubrication. Special calculation methods (e. g. FEM ) can be used for critical basic units and selected problems.

#### Sealings

The packing of the shaft outlets is realized by means of non-contacting labyrinth seals.

#### **Additional Equipment**

Additional equipment such as couplings, coupling claddings, rotor barring gears, soft starters, pumps, lube systems etc. belong to the scope of delivery on costumer's request.

#### **Test Stand Running**

On the test stand there can be simulated conditions similar to the operation and is ensured an extensive control of all component including acquisition of measuring values. The general representation of the gear unit is not strictly binding. The weights given in the tables are averages (without oil filling) and are subject to the technical development. This also refers to the dimensions.

#### **Main Dimensions**





BG	i*	a	D1	11	D2	12	L1	L2	L3	L4	L5	L6	B1	B2	B3
	1		80	170	90	170	630	250			250	550	535		105
160	2	160	45	110	80	170	570	190	-	-	190	500	380	-	85
	3		28	60	60	140	555	175			175	480	320		75
	1		90	170	100	210	750	270			270	620	580		110
200	2	200	55	110	80	170	700	220	-	-	220	600	490	-	85
	3		40	110	65	140	640	180			180	560	350		80
	1		100	210	110	210	850	290			290	700	630		125
225	2	225	65	140	100	210	765	235	-	-	235	650	530	-	125
	3		40	110	80	170	720	200			200	615	430		105
	1		110	210	130	250	880	300			300	780	740		150
250	2	250	90	170	110	210	820	240	-	-	240	720	620	-	150
	3		55	110	90	170	795	215			215	690	520		140
	1		140	250	160	300	910	315			315	8800	750		135
280	2	280	85	170	125	210	880	255	-	-	255	740	630	-	135
	3		50	110	100	201	855	220			220	715	560		135
	1		140	250	160	300	1,005	340			340	900	850		170
315	2	315	110	210	140	250	960	265	-	-	265	820	650	-	135
	3		65	140	125	210	945	230			230	810	600		135
	1		160	300	180	300	1,085	360			360	950	850		155
355	2	355	100	210	160	300	1,040	270	-	-	270	890	680	-	130
	3		65	140	130	250	1,025	240			240	845	600		130
	1		180	300	220	350	1,210	380			380	1,120	1,020		220
400	2	400	110	210	180	300	1,160	290	-	-	290	1,030	760	-	150
	3		70	140	140	250	1,140	245			245	990	630		125
	1		200	350	240	410	1,300	415			415	1,180	1,140		235
450	2	450	125	210	210	350	1,280	310	-	-	310	1,110	850	-	170
	3		80	170	160	300	1,250	260			260	1,050	730		170
	1		210	350	250	410	1,470	430			430	1,270	1,180		235
500	2	500	140	250	220	350	1,400	320	-	-	320	1,180	860	-	170
	3		90	170	200	350	1,370	265			265	1,170	710		150
	1		220	350	280	470	1,575	465		1,110	465	1,450	1,300	830	150
560	2	560	160	300	250	410	1,510	340	175	950	340	1,300	940	620	100
	3		100	210	200	350	1,485	275		950	275	1,300	770	500	100
	1		240	410	320	470	1,810	520		1,300	520	1,620	1,400	915	200
630	2	630	160	300	280	470	1,720	380	160	1,240	380	1,560	1,040	715	120
	3		100	210	210	350	1,680	330		1,210	330	1,530	880	580	120
	1		280	470	340	550	2,020	610		1,600	610	1,900	1,530	980	200
710	2	710	220	350	320	470	1,920	430	150	1,540	430	1,840	1,300	780	170
	3		125	210	250	410	1,870	350		1,350	350	1,650	930	580	130
	1		280	470	380	550	2,200	625		1,770	625	2,000	1,620	1,105	300
800	2	800	210	350	360	550	2,115	455	115	1,720	455	1,950	1,330	890	260
;	3	500	125	210	260	410	1,050	370		1,660	370	1,890	960	660	120

Note: D1, I1, D2, I2 are maximum values which are matched to the couplings used. Shaft ends according to DIN 6885.

BG	i*	a	<b>B</b> 4	B5	H1	H2	H3	H4	<b>S</b> 1	S2	<b>S</b> 3	<b>S</b> 4	\$5	D3	m (t)		
	1		280	280									490	10	0.4		
160	2	160	205	205	500	250	25	-	60	-	200	-	335	18	0.3		
	3		175	175									290	14	0.2		
	1		330	330									520	00	0.6		
200	2	200	260	260	590	300	32	-	75	75	210	-	430	22	0.5		
	3		195	195									300	18	0.3		
	1		560	560									570	00	0.8		
225	2	225	280	280	650	315	36	-	75	75	260	-	470	22	0.5		
	3		235	235									380	18	0.4		
	1		410	410									690		1		
250	2	250	310	310	715	355	40	-	75	75	300	-	570	22	0.8		
	3		245	245									490		0.6		
	1		415	415									690		1.4		
280	2	280	330	330	770	400	45	-	90	90	280	-	570	26	1		
	3		270	270									500		0.8		
	1		468	468									790		2		
315	2	315	365	365	850	450	50	-	90	90	340	-	590	26	1.5		
	3		290	290									540		1		
	1		465	465									780		2.3		
355	2	355	365	365	950	500	55	-	100	100	375	-	610	33	1.6		
	3		300	300									530		1.3		
	1		580	585									970		3.4		
400	2	400	440	440	1,025	525	60	-	130	130	450	-	680	33	2.3		
	3		355	355											550		1.8
	1		630	630									1,050		4.7		
450	2	450	460	460	1,100	560	65	-	120	120	450	-	760	39	3.1		
	3		365	365									640		2.3		
	1		650	650									1,090		4.7		
500	2	500	490	490	1,200	630	70	-	130	130	475	-	770	39	4.1		
	3		395	395									620		3.2		
	1		710	710							450	425	1,200		7.1		
560	2	560	560	560	1,150	500	70	250	150	150	425	300	840	39	5.5		
	3		415	415							480	235	670		3.7		
	1		760	760							500	520	1,300		9.6		
630	2	630	580	580	1,260	560	75	250	140	140	580	380	940	45	7		
	3		450	450							620	320	780		5		
	1		825	825							700	700	1,430		13.2		
710	2	710	710	710	1,350	560	80	300	150	150	800	500	1,200	45	12		
	3		495	495							650	350	830	10	6.7		
	1		880	800							640	640 1,	1,520	)	16.1		
800	2	800	735	735	1,470	1,470 630	90	300	160	0 160	730	450	) 1,230 45	45	14		
800	3		530	530							760	360	880		8.2		

Dimensions are subject to the technical development. Ratio range: i\*=1 for 1.10 i 2.8; i\*=2 for 2.8 i 5.6; i\*=3 for 5.6 i 8.0







P - Nominal power of the driving machinery (kW)

n<sub>1</sub> - Speed of the high-speed shaft (min<sup>-1</sup>)

 ${\rm n_2}$  - Speed of the low-speed shaft (min<sup>-1</sup>)

ai	1.12	2.8	3.15	5.6	6.3	8
160	19.1			20		
200	19.1			20		
225	17.3	19.1		2	0	
250	13.6	17.3	19.1		20	
280	11.2	15.2	19.1		20	
315	10.8	13.6	15.2		20	
355	10.6	13	14		20	
400	9.5	10.6	12.5	17.3	2	0
450	8.5	9.5	11.9	16.2	19.1	20
500	8	9	10	15.2	17.3	20
560	7.6	8.5	9.5	13.6	15.2	19.1
630	7.2	8	9.5	11.9	13.6	15.2
710	6.3	7	7.6	10.8	11.9	13.5
800	5.8	6.5	7	9.5	10.5	11.5

Maximum pemissible speeds on the high-speed shaft  $n_1$  (1,000 min<sup>-1</sup>)

#### Remark:

Power diagramm and speed table apply to the standard design and are useful for the preselection of the gearing size. In case of selecting an optimum size, of critical conditions, and of speed exeeding we ask for your inquiry.

Example for Ordering: Turbo-Helical Gear Unit with single-helical gearing Size 355 Ratio i = 3.15 **TSG 1-355x3.15** 



	A STATE	
		Gear drive MT300Y02
Charles I	Motor rating P (kW)	3,000
	Motor speed n <sub>1</sub> (rpm)	8,545.5
	Transmission ratio i	2.152
	Output speed n <sub>2</sub> (rpm)	18,385.70
	Output torque (kNm)	1.56
	Bearings fitted	plain journal bearings
	Weight (kg)	approx. 900
	Used for	oil and gas industry

The Contraction of the second se	Gear drive MT600A03
Motor rating P (kW)	6,000
Motor speed n <sub>1</sub> (rpm)	3,000
Transmission ratio i	2.848
Output speed n <sub>2</sub> (rpm)	8,545,50
Output torque (kNm)	6.71
Bearings fitted	plain journal bearings
Weight (kg)	approx. 2,500
Used for	oil and gas industry

#### Special gear drive solutions

	MT575B04	MT18MB03	MT16MB07	MT050A01	MT300Y02	MT600A03
Nominal power P (kW)	5,750	18,200	15,700	500	3,000	6,000
Transmission ratio i	4.353	2.641	6.515	1.38	2.15	2.85
Revolutions at Input shaft $n_1$ (min <sup>-1</sup> )	1,450	1,500	1,500	2,975	8,546	3,000
Revolutions at Output shaft n <sub>2</sub> (min <sup>-1</sup> )	6,312	3,962	9,773	4,106	18,386	8,546
Service factor K <sub>A</sub>	1.59	1.66	1.71	-	1.8	1.8
Motor rating P/n,	3.966	12.133	10.467	0.168	0.351	2



Example:

The gear drives MT300Y02 and MT600A03 are an alternative set of gearboxes for compressor stations with the following technical characteristics:

Mass capacity kg/f: 9.53 (9.73)

Volume capacity at suction conditions m<sup>3</sup>/min: 355 (345)

Final gas pressure MPa: 3.82

Gas density kg/m<sup>3</sup>: 0.83 (0.913)



## Turbo-Planetary Gear Units

References: more than 2,000 units

<image/>		
		Planetary gear drive TPG (HUG) - type A/B
	Size	16 up to 50 (8 sizes available)
	Motor rating P (kW)	1,600 up to 12,500
	Motor speed n <sub>1</sub> (rpm)	276 up to 6,300
	Output speed n <sub>2</sub> (rpm)	1,500 up to 20,000
	Bearings fitted	plain journal bearings
	Used for	power generation plant, steam turbine



#### **General Specification**

The turbo-planetary gear unit are produced as types A and B in eight sizes. The type A is provided with rotaring planet carrier, the output shaft and the drive shaft have the same senses of rotation. In the case of the type - provided with fixed planet carrier the output shaft and the drive shaft have opposite senses of rotation. The two types may be used for both the increase and the reduction of the speed. These single-stage turbo-planetary gear unit are mounted, above all in the driving units of centrifugal compressors, generators, turbines and pumps.

#### Gear Units

The erection of the gear units can be realized by centre suspension as well as by means of a food-mounted design. On the gear side of the slow-speed shaft there can be mounted auxiliary drives, e.g. for speedometers, controlle and pumps. The gear unit should be mounted on a rugged, flat, torsionally stiff foundation which cannot be displaced, the minimum natural frequency of which exceeds considerably the maximum forcing frequency of the gear unit. If the gear unit shall be mounted on a deep foundation, the manufacturing enterprise will inform, on special request, about its functional forcing frequency.

#### Housing

The casings are made of cast iron. The housings are so designed as to minimize vibrations and to ensure a silent running. All housings are equipped with lifting facilities and mounting surfaces for alignment.

#### Gearing

All the power transmission elements of the gear unit have been made of case-hardened and heat-treated steels. The tooth profiles of the sun pinions and the planet wheels are protuberance toothed, case-hardened and ground. The sun gear and the internally toothed coupling parts are slotted and nitrided.

#### Bearing

The planet wheels and the shafts are running in special sliding bearings. All bearing points consist of cast white metal.

#### Lubrication

The tooth action points and the bearings are lubricated with the help of a pressure oil circulation system. On special request, one ore two oil pumps can be mounted to the gear unit side where the slowly running shaft is installed. Necssary oil pressure at the oil inlet of the reducer: 0.25 MPa for minimum

Oil inlet temperature: 40 °C up to 45 °C

Maximum oil outlet temperature: 80 °C

The reducers should be equipped with a collecting tank for the lubricating oil and a heat exchanger for cooling. For lubrication an oil of high resistance to pressure and ageing should be used offering the properties to follow.

#### Shafts

Shafts are made of steel hardened and tempered. At the high-speed shaft the turbo-planetary gear unit must only be coupled with machines of which the radial bearing of their shafts is well defined. The high speed shaft end d<sub>1</sub> cannot be loaded by radial forces. Axial movement resulting from the thermal expansion are taken up. The axial movement must not exceed 3.5 mm. Additional external forces are inadmissible. The slowly running shaft d<sub>2</sub> cannot be loaded by radial forces with the exception of such ones resulting from alignment tolerances. The alignment error must not exceed the value stated in the operating instructions.

#### Power Capacity Characteristics for the Determination of the size

The power capacity characteristics of Diagrams 1 and 2 for the power capacity/speed ratio on the shaft  $d_1$  are valid for uninterrupted, shockless continous operation.

During starting the twofold torque may be transmitted.

In case of varying operational conditions as well as when operating in the part load range the manufacturing enterprise should be consulted. When using synchronous motors as driving units the manufacturing enterprise should be consulted with regard to the starting operation.





Maximum admissible speed of the shaft d2:

 $n_2 = 1,800$  rpm for the sizes A 16 up to A 36 and AM up to AM 36

- $n_2 = 1,800$  rpm for the sizes A 40 and AM 40 i = 6
- $\rm n_{_2}$  =1,500 rpm for the sizes A 40 and AM 40 i = 6

 $\rm n_{_2}$  =1,500 rpm for the sizes A 45 and A 40, AM 45 and AM 50

Higher speeds on request.

The transmission ratios 3.15; 4; 5; 6.3; 8; 10; 11.2 are nominal transmission ratios with a tolerance of  $\pm 3\%$ Other transmission ratios may be agreed upon with the manufacturing enterprise.

Example: P = 4,500 kW; n<sub>1</sub> = 7,500 rpm.; n<sub>2</sub> = 1,500 rpm.; size A 40 or AM 40



Maximum admissible speed of the shaft d2:

 $\rm n_{_2}$  = 3,600 rpm for the sizes B 16 up to B 40 and BM 16 up to BM 40

 $\rm n_{_2}$  =1,800 rpm for the sizes B 45 up to B 50 as well as BM 45 and BM 50

Higher speeds on request.

The transmission ratios 2.12; 2.5; 3.15; 4; 5; 6.3; 8; 10 are nominal transmission ratios with a tolerance of  $\pm 3\%$ Other transmission ratios may be agreed upon with the manufacturing enterprise.

Example: P = 3,000 kW; n<sub>1</sub> = 13,000 rpm.; n<sub>2</sub> = 3,000 rpm.; size B 25 or BM 25







Siz	es																	Weig	ht kg
А	В	b,	b <sub>2</sub>	d <sub>3</sub>	d <sub>4</sub>	d <sub>5</sub> ,	e <sub>1</sub>	e <sub>2</sub>	e <sub>3</sub>	e <sub>4</sub>	e <sub>5</sub>	e <sub>6</sub>	h	h <sub>2</sub>	h <sub>3</sub>	l <sub>3</sub>	l <sub>4</sub>	А	В
						9 <sub>6</sub>							-1.0						
16	-	600	300	22	16	240	140	540	340	625	315	60	265	586	35	350	70	380	-
20	16	650	350	22	16	240	165	580	420	705	320	60	280	626	40	400	70	490	410
25	20	740	440	22	16	220	195	670	447	887	400	60	315	656	45	460	70	680	650
32	25	820	500	26	16	220	220	740	475	955	490	60	355	715	50	520	70	880	840
36	32	900	540	33	25	360	255	810	472	975	510	75	400	815	55	600	800	1.250	1.050
40	36	1,000	600	33	25	360	270	900	560	1,030	520	75	450	929	60	640	90	1,700	1,450
45	40	1,100	660	33	25	360	300	1,000	635	1,203	600	75	500	989	60	700	90	2,400	2,200
50	45	1,150	700	39	25	400	325	1,050	700	1,350	650	75	530	1,039	65	750	100	3,000	2,700
-	50	1,280	800	39	25	400	340	1,160	760	1,510	700	75	560	1,138	65	800	100	-	3,450

d<sub>4</sub> - Taper pins required according to DIN-7977; 2 predrilled holes for taper pins, diagonally opposed

 $d_{_5}$ ,  $e_{_5}$ ,  $l_{_4}$  - Variations in the stated dimensions may be agreed upon with the manufacturing enterprise

Weight - without oil pumps and without flange execution (F); recommended values for average nominal transmission ratios (Dimensions in mm)





Siz	zes												Weig	Weight kg	
AM	ВМ	b <sub>3</sub>	b <sub>4</sub>	e <sub>7</sub>	e <sub>8</sub>	e <sub>9</sub>	e <sub>10</sub>	h <sub>4</sub>	h <sub>s</sub>	h <sub>6</sub>	l <sub>7</sub>	l <sub>5</sub>	AM	BM	
									-0.5						
16	-	660	560	600	35	105	50	50	60	256	586	273	380	-	
20	16	700	600	640	47.5	120	60	50	60	280	626	305	490	410	
25	20	800	700	740	42.5	142.5	60	50	60	315	656	355	680	650	
32	25	860	740	790	50	160	60	60	71	350	740	390	880	840	
36	32	950	825	870	57.5	182.5	75	60	71	400	815	445	1,250	1,050	
40	36	1,100	900	1,000	50	185	80	70	80	450	929	470	1,700	1,450	
45	40	1,160	1,020	1,080	72	225	75	70	80	500	989	530	2,400	2,200	
50	45	1,250	1,100	1,170	40	240	80	70	80	550	1,059	600	3,000	2,700	
-	50	1,320	1,160	1,240	55	260	80	70	80	575	1,153	620	-	3,450	

Weight - without oil pumps and without flange execution (F); recommended values for average nominal transmission ratios (Dimensions in mm)



#### Flange dimensions (F) of the shaft $d_2$

Siz	es									
A; AM	B; BM	input d <sub>6</sub>		d <sub>7</sub>	d <sub>7</sub> d <sub>8</sub>		h <sub>8</sub>	h <sub>9</sub>	k	Weight increas
		speed		H <sub>8</sub>					±0.05	kg
36	32	1,500	380	140	38.5	700	60	12	300	49
40	36	1,500	420	140	38.5	700	60	12	330	56
-	40	3,000	380	140	38.5	755	60	12	300	49
45	-	1,500	420	140	38.5	755	60	12	330	56



#### Dimension of the shaft ends

Sizes	Shaft end with 2 fitting keys,		Hub hole with 2 hub slots,					
	staggered for 180°		staggered for 180°					
	i = 2.12 up to 11.2		i = 2.12 up to 4		i > 4 up to 6.3		i > 6.3 up to 11.2	
	d <sub>2</sub>	l <sub>2</sub>	d,	Ι,	d,	I,	d,	I,
	m6		H6		H6		H6	
16	90	130	70	105	60	105	-	-
20	100	165	80	130	70	105	-	-
25	110	165	90	130	80	130	60	105
32	125	165	100	165	90	130	70	105
36	140	200	110	165	100	165	80	130
40	160	240	125	165	110	165	90	130
45	180	240	140	200	125	165	100	165
50	200	280	140	200	140	200	110	165

Shaft ends / Hub hole - See the main dimensions; Fitting keys, dimensions according to i DIN 6885 Hub slot width: tolerance zone J9 or P9, state, please, in case of order (J) or (P)



#### **Delivery program**

## Gear units for conveying plants and open-cast mining equipments

- bucket wheel gear units
- bucket dredger gear units
- slewing gear units
- crawler gear units
- belt conveyor gear units
- crane gear units
- hoisting gear units
- grab hoisting gear units

#### Gear units for cement plants

- mill drives
- kiln drives
- standard helical- and bevel-helical-planetary gear units

## Gear units for equipments of the rolling mill and metallurgical industry

- pinion stands
- distributor gear units
- table roller drives
- reducers for stretch levelling plants

#### Gear units for chemical processing and energy plants

- agitators gear units
- calenders gear units
- gear units for kneaders
- gear units for injection moulding machines
- turbo-planetary gear units
- turbo-helical gear units

## Gear units for marine propulsion engines and marine equipment

- for cargo vessels
- for catcher and full-freezer trawlers
- for sea-going and harbour tugs
- for passenger ships
- multi-speed marine reversing gear units
- gear units for trawl warp winch

#### Special gear units



## www.geartecag.com



### The GearTec Group:



MDM Mega Drive Magdeburg GmbH Grabower Straße 6 39126 Magdeburg Germany

Phone +49 (0) 391 99 00 05-0 Fax +49 (0) 391 99 00 05-99 info@megadrive-md.de www.megadrive-md.de



GTD Getriebetechnik Dessau GmbH Thomas-Müntzer-Straße 42 06842 Dessau Germany

Phone +49 (0) 340 83 22 10 Fax +49 (0) 340 83 25 85 contact@gt-dessau.de www.gt-dessau.de



GTM Getriebetechnik Magdeburg GmbH Steinfeld Str. 14 39179 Barleben Germany

Phone +49 (0) 39203 517-0 Fax +49 (0) 39203 517-199 info@getriebetech.de www.getriebetech.de